

NEW YORK UNIVERSITY
INSTITUTE OF MATHEMATICAL SCIENCES
LAWRENCE
25 Waverly Place, New York 3, N. Y.

IMM-NYU 281
FEBRUARY 1961



NEW YORK UNIVERSITY
INSTITUTE OF
MATHEMATICAL SCIENCES

Extension of A Thick Infinite Plate With A Circular Hole

EDWARD L. REISS

PREPARED UNDER
CONTRACT NO. NONR-285 (42)
WITH THE
OFFICE OF NAVAL RESEARCH
UNITED STATES NAVY

REPRODUCTION IN WHOLE OR IN PART
IS PERMITTED FOR INTERNAL USE
OF THE UNITED STATES GOVERNMENT.

IMM-281
e.1

IMM-NYU 281
February 1961

New York University
Institute of Mathematical Sciences

EXTENSION OF A THICK INFINITE PLATE WITH
A CIRCULAR HOLE

Edward L. Reiss

This report represents results obtained at the
Institute of Mathematical Sciences, New York
University, under the sponsorship of the Office
of Naval Research, United States Navy, Contract
No. Nonr-285(42).

1961

1 Introduction.

The plane stress solution [1] is conventionally employed to estimate the stress concentration due to a circular hole in a plate that is uniaxially stretched at infinity. However, this solution is not a solution of the exact theory.¹ Nevertheless, we expect it to yield an accurate approximation if $\epsilon = \frac{h}{R}$ is sufficiently "small".² We also expect that the accuracy will increase as $\epsilon \rightarrow 0$.

The precise relationship between the plane stress and the exact theories is given in [3] for simply connected plates with "smooth" boundary curves. In addition, a boundary layer procedure³ is outlined for obtaining increasingly accurate approximations to the solution of the exact theory. These approximations are given as "three-dimensional corrections" to the plane stress solution.

In this paper we extend the method of [3] to our stress concentration problem. Results are obtained in the form of a power series in ϵ . We give here only terms up to and including

¹ We refer to the three-dimensional linear theory of elasticity for homogeneous and isotropic materials as the exact theory. There are some special cases for which the plane stress solution is a solution of the exact theory [2].

² Here, h is one half of the plate thickness and R is the radius of the hole.

³ It is a generalization of the one given by Friedrichs [4] and Friedrichs and Dressler [5] in a study of the "bending" of plates. See also [6] and [7].

second order. Within this approximation we show that the plane stress theory yields extremely accurate, although non-conservative predictions of the maximum stress concentration for "small" but finite values of ϵ . This accuracy depends upon Poisson's ratio ν . For example, with $\nu = 1/3$ the error in the plane stress solution is less than 5% if $\epsilon \leq .3$. For "larger" values of ν and ϵ , the error increases. More accurate approximations to the solution of the exact theory, which may be necessary for these values of ϵ , can be obtained by determining third and higher terms in the expansion.

Other approximate solutions of the exact theory for this problem are given by Sternberg and Sadowsky [8] using a modification of the Ritz method and Green [9] and Alblas [10] who employed infinite series expansions.⁴ Our results for the stress concentration compare favorably with those of Alblas if $\epsilon \leq \frac{1}{4}$, see Figs. 1 and 2. Agreement is especially good with his "asymptotic" solution which is closely related to our approximation method.

2. Formulation.

We introduce a cylindrical coordinate system r, θ, z . An infinite plate of thickness $2h$ with a circular hole of radius R is considered as a three-dimensional elastic body bounded by the planes (the "faces" of the plate) $z = \pm h$ and the cylindrical surface $r = R$. The origin of the coordinates is fixed at the center of the hole on the midplane of the plate, $z = 0$. The plate

⁴ See also Reissner [11].

is stretched at "infinity" by a constant tensile force T in a fixed direction which we take as the x -axis. The boundary of the hole, i.e. the edge, and the faces of the plate are free of forces.

If we introduce the dimensionless variables:

$$\xi = \frac{r-R}{R} , \quad \xi \geq 0 ; \quad \zeta = \frac{z}{h} , \quad |\zeta| \leq 1 ,$$

and the parameter,

$$\varepsilon = \frac{h}{R} ,$$

then the faces of the plate are given by $\zeta = \pm 1$ and the boundary of the hole by $\xi = 0$.

Considering the components of stress as functions of ξ, θ, ζ and employing an obvious notation, the stress formulation of the exact theory is given by:

Equilibrium Equations,

$$(1) \quad \begin{aligned} \tau_{rz,\zeta} + \varepsilon [\sigma_{r,\xi} + (1+\xi)^{-1} (\sigma_{r\theta,\theta} + \sigma_r - \sigma_\theta)] &= 0 , \\ \sigma_{\theta z,\zeta} + \varepsilon [\sigma_{r\theta,\xi} + (1+\xi)^{-1} (\sigma_{\theta,\theta} + \sigma_{r\theta})] &= 0 , \\ \sigma_{z,\zeta} + \varepsilon [\sigma_{rz,\xi} + (1+\xi)^{-1} (\sigma_{\theta z,\theta} + \sigma_{rz})] &= 0 ; \end{aligned}$$

$$\begin{aligned}
 \sigma_z, \xi\xi + \Omega, \xi\xi &= -\varepsilon^2 \Delta \sigma_z , \\
 \sigma_{rz}, \xi\xi &= -\varepsilon \Omega, \xi\xi + \varepsilon^2 [-\Delta \sigma_{rz} + (1+\xi)^{-2} (\sigma_{rz} + 2\sigma_{\theta z, \theta})] , \\
 (2) \quad \sigma_{\theta z}, \xi\xi &= -\varepsilon \Omega, \theta \xi + \varepsilon^2 [-\Delta \sigma_{\theta z} + (1+\xi)^{-2} (\sigma_{\theta z} - 2\sigma_{rz, \theta})] , \\
 \sigma_r, \xi\xi &= \varepsilon^2 A , \quad \sigma_\theta, \xi\xi = \varepsilon^2 B , \quad \sigma_{r\theta}, \xi\xi = \varepsilon^2 C .
 \end{aligned}$$

Here,

$$\begin{aligned}
 (3a) \quad A(\xi, \theta, \xi; \varepsilon) &= -\Delta \sigma_r - \Omega, \xi\xi + 2(1+\xi)^{-2} (\sigma_r - \sigma_\theta + 2\sigma_{r\theta, \theta}) , \\
 (3b) \quad B(\xi, \theta, \xi; \varepsilon) &= -\Delta \sigma_\theta - (1+\xi)^{-1} \Omega, \xi + 2(1+\xi)^{-2} (-\Omega, \theta \theta + 2\sigma_\theta - 2\sigma_r - 4\sigma_{r\theta, \theta}) , \\
 (3c) \quad C(\xi, \theta, \xi; \varepsilon) &= -\Delta \sigma_{r\theta} - (1+\xi)^{-1} \Omega, \xi \theta + (1+\xi)^{-2} (\Omega, \theta + 4\sigma_{r\theta} + \sigma_\theta, \theta - \sigma_r, \theta) ,
 \end{aligned}$$

$$\Delta \sigma = \sigma_{\xi\xi} + (1+\xi)^{-1} \sigma_\xi + (1+\xi)^{-2} \sigma_{\theta\theta} , \quad \Omega = \frac{1}{1+v} (\sigma_r + \sigma_\theta + \sigma_z) ,$$

v is Poisson's ratio and a comma indicates partial differentiation
To complete the formulation we require appropriate boundary
conditions. These are obtained by specifying the applied forces
as:

$$(4a) \quad \sigma_{rz}(\xi, \theta, \pm 1) = \sigma_{\theta z}(\xi, \theta, \pm 1) = \sigma_z(\xi, \theta, \pm 1) = 0 ;$$

$$(4b) \quad \sigma_{rz}(0, \theta, \xi) = \sigma_r(0, \theta, \xi) = \sigma_{r\theta}(0, \theta, \xi) = 0 ;$$

$$(4c) \quad \sigma_{rz}(\infty, \theta, \xi) = 0 , \quad \sigma_r(\infty, \theta, \xi) = T \cos^2 \theta , \quad \sigma_{r\theta}(\infty, \theta, \xi) = -\frac{1}{2} T \sin 2\theta ,$$

where T is the constant tensile force at infinity in the x -direction.

Since we are concerned only with the extension of plates, it can be shown [5], without loss of generality, that σ_r , τ_θ , σ_z and $\tau_{r\theta}$ are even functions of ξ while τ_{rz} and $\tau_{\theta z}$ are odd functions of ξ . In the following sections we shall make frequent use of these properties without explicit reference.

3. The Interior Problems.

We assume that each stress component, indicated by the generic symbol $\sigma(\xi, \theta, \zeta; \varepsilon)$, can be asymptotically represented by a power series in ε :

$$(5) \quad \sigma(\xi, \theta, \zeta; \varepsilon) \sim \sum_{n=0}^{\infty} \sigma^n(\xi, \theta, \zeta) \varepsilon^n .$$

Here σ^n are called the interior stress coefficients and we define $\sigma^n = 0$ if $n < 0$. We assume that the σ^n possess the same even or odd property as σ . Substituting these expansions into (1-3) and (4a) and equating coefficients of like powers of ε yields a system of differential equations that are satisfied by the σ^n . The analysis of this system is elementary and similar to that outlined in [3]. Therefore, the calculations are not explicitly exhibited. Instead some of the results are listed below. For example, we can show that:

$$(6) \quad \sigma_{rz}^n = \sigma_{\theta z}^n = \sigma_z^n = 0 , \quad \Omega^n(\xi, \theta, \zeta) = \Omega^n(\xi, \theta) , \quad \left. \right\} n=0,1,\dots$$

$$(7) \quad \left. \begin{array}{l} \sigma_{r,\xi}^n + (1+\xi)^{-1}(\sigma_{r\theta,\theta}^n + \sigma_r^n - \sigma_\theta^n) = 0 , \\ \sigma_{r\theta,\xi}^n + (1+\xi)^{-1}(\sigma_{\theta,\theta}^n + \sigma_{r\theta}^n) = 0 , \quad \Delta\Omega^n = 0 ; \end{array} \right\} n=0,1,\dots$$



$$(8a) \quad \sigma_r^n = S_r^n(\xi, \theta), \quad \sigma_\theta^n = S_\theta^n(\xi, \theta), \quad \sigma_{r\theta}^n = S_{r\theta}^n(\xi, \theta), \quad n=0,1;$$

$$\sigma_r^n = \frac{1}{2} A^{n-2}(\xi, \theta) \zeta^2 + S_r^n(\xi, \theta), \quad \sigma_\theta^n = \frac{1}{2} B^{n-2}(\xi, \theta) \zeta^2 + S_\theta^n(\xi, \theta);$$

(8b)

$$\sigma_{r\theta}^n = \frac{1}{2} C^{n-2} \zeta^2 + S_{r\theta}^n(\xi, \theta), \quad n=2,3,$$

where A^n , B^n and C^n are obtained from (3) using (5).

We define the m -th order interior problem (Problem I^m) as the boundary value problem that contains (6-8) with $n = m \leq 3$ and appropriate boundary conditions which are obtained in the following sections. Equations (6-8) with $n = 0$ are, in our notation, the stress relations and the differential equations of the classical theory of plane stress in polar coordinates [1].

4. Formulation of the Boundary Layer Problem.

The results given in the previous section are obtained without reference to the edge boundary conditions (4b). In fact, the expansions (5) cannot, in general, satisfy these conditions. If they did, it then follows from (8) and (3) that more boundary conditions then are appropriate for the solution of (7) are specified. Thus, if the series (5) represent the solution of the exact theory they do so in a region away from the edge which we call the "interior domain". The region of the plate adjacent to and including the edge where (5) may deviate rapidly from the solution of the exact theory is called the "boundary layer". To obtain expansions that may converge uniformly up to and including the edge we assume, as in [3]^{*}, that the deviation of (5) from

* See also [6,7].

the exact solution occurs only in the direction normal to the edge, i.e. in the ξ direction. We then introduce the "stretched" boundary layer variable [12] η as:

$$(9) \quad \eta = \frac{\xi}{\varepsilon} .$$

Considering the η, θ, ζ coordinate system, boundary layer stresses indicated by the generic symbol $f(\eta, \theta, \zeta; \varepsilon)$ are defined as:

$$(10) \quad f(\eta, \theta, \zeta; \varepsilon) = \sigma(\xi, \theta, \zeta; \varepsilon) .$$

To determine approximations to the exact stress distribution near the edge of the hole and appropriate boundary conditions for the interior problems we assume that:

$$(11) \quad f(\eta, \theta, \zeta; \varepsilon) \sim \sum_{n=0}^{\infty} f^n(\eta, \theta, \zeta) \varepsilon^n .$$

The f^n are called the boundary layer stress coefficients and we set $f^n \equiv 0$ if $n < 0$. We further assume that each f^n possesses the same even or odd property as the corresponding f .

Substituting (9-10) into the exact theory (1-4) and equating coefficients of like powers of ε we find from the coefficients of ε^n that the f^n satisfy:

$$(12) \quad \begin{aligned} f_{rz, \zeta}^n + f_{r, \eta}^n + \sum \left(f_{r\theta, \theta}^1 + f_r^1 - f_\theta^1 \right) &= 0 , \\ f_z^n + f_{rz, \eta}^n + \sum \left(f_{\theta z, \theta}^1 + f_{rz}^1 \right) &= 0 , \\ f_{\theta z, \zeta}^n + f_{r\theta, \eta}^n + \sum \left(f_{\theta \theta, \theta}^1 + 2f_{r\theta}^1 \right) &= 0 ; \end{aligned}$$

$$\begin{aligned}
 & \nabla^2 f_r^n + \Gamma_{\eta\eta}^n + \sum' f_{r,\eta}^i + \sum'' \left[f_{r,\theta\theta}^i - 2(f_r^i - f_\theta^i + 2f_{r\theta,\theta}^i) \right] = 0 , \\
 & \nabla^2 f_z^n + \Gamma_{\zeta\zeta}^n + \sum' f_{z,\eta}^i + \sum'' f_{z,\theta\theta}^i = 0 , \\
 (13) \quad & \nabla^2 f_\theta^n + \sum' (f_{\theta,\eta}^i + \Gamma_{\eta\theta}^i) + \sum'' \left[2(f_r^i - f_\theta^i + 2f_{r\theta,\theta}^i) + f_{\theta,\theta\theta}^i + \Gamma_{\theta\theta}^i \right] = 0 , \\
 & \nabla^2 f_{rz}^n + \Gamma_{\eta\zeta}^n + \sum' f_{rz,\eta}^i + \sum'' (f_{rz,\theta\theta}^i - f_{rz,\theta}^i - 2f_{\theta z,\theta}^i) = 0 , \\
 & \nabla^2 f_{r\theta}^n + \sum' (f_{r\theta,\eta}^i + \Gamma_{\eta\theta}^i) + \sum'' \left[f_{r\theta,\theta\theta}^i + 2\frac{\partial}{\partial\theta}(f_r^i - f_\theta^i) - 4f_{r\theta,\theta}^i - \Gamma_{\theta\theta}^i \right] = 0 , \\
 & \nabla^2 f_{\theta z}^n + \sum' (f_{\theta z,\eta}^i + \Gamma_{\eta\zeta}^i) + \sum'' \left[f_{\theta z,\theta\theta}^i - f_{\theta z,\theta}^i + 2f_{rz,\theta}^i \right] = 0 ;
 \end{aligned}$$

$$\begin{aligned}
 f_{rz}^n(\eta, \theta, \pm 1) &= f_z^n(\eta, \theta, \pm 1) = f_{\theta z}^n(\eta, \theta, \pm 1) = 0 , \\
 f_{rz}^n(0, \theta, \zeta) &= f_r^n(0, \theta, \zeta) = f_{r\theta}^n(0, \theta, \zeta) = 0 , \\
 (14) \quad f_{rz}^n(\infty, \theta, \zeta) &= 0 , \quad f_r^n(\infty, \theta, \zeta) = \begin{cases} T \cos^2 \theta, & n = 0 \\ 0, & n \neq 0 \end{cases} . \\
 f_{r\theta}^n(\infty, \theta, \zeta) &= \begin{cases} -T/2 \sin 2\theta, & n = 0 \\ 0, & n \neq 0 \end{cases} .
 \end{aligned}$$

Here,

$$\Gamma^n = \frac{1}{1+\nu} (f_r^n + f_\theta^n + f_z^n) , \quad \nabla^2 \equiv \frac{\partial^2}{\partial\eta^2} + \frac{\partial^2}{\partial\zeta^2} ,$$

$$\sum' A^i = \sum_{i+j+l=n} (-1)^j \eta^j A^i , \quad \sum'' A^i = \sum_{i+j+2=n} (-1)^j (j+1) \eta^j A^i .$$

In addition we require the boundary layer stresses to approach or "match" the interior stresses as $\varepsilon \rightarrow 0$. Specifically, we assume that each $\sigma^n(\xi, \theta, \zeta)$ has, near $\xi = 0$, a Taylor series expansion in ξ :

$$\sigma^n(\xi, \theta, \zeta) = \sum_{m=0}^{\infty} s_m^n(\theta, \zeta) \xi^m ,$$

where

$$(15a) \quad s_m^n = \frac{1}{m!} \frac{\partial^m \sigma^n(0, \theta, \zeta)}{\partial \xi^m} , \quad n = 0, 1, \dots .$$

Therefore from (5) and (9) we have, in some region about $\xi = 0$,

$$(16) \quad \sigma(\xi, \theta, \zeta; \varepsilon) \sim \sum_{n=0}^{\infty} \sigma^{*n}(\eta, \theta, \zeta) \varepsilon^n ,$$

where

$$(15b) \quad \sigma^{*n}(\eta, \theta, \zeta) = \sum_{m=0}^n s_m^{n-m}(\theta, \zeta) \eta^m$$

are the interior coefficients near $\xi = 0$ as functions η , θ and ζ . For each n we define the "reduced boundary layer stress coefficients", $F^n(\eta, \theta, \zeta)$ as:

$$(17) \quad F^n(\eta, \theta, \zeta) \equiv f^n(\eta, \theta, \zeta) - \sigma^{*n}(\eta, \theta, \zeta) .$$

The "matching condition" or the asymptotic form for the f^n is obtained from (9), (10) and (16) by associating each f^n with the corresponding σ^{*n} as $\varepsilon \rightarrow 0$. Using (9) and (17) we write this condition as:[†]

$$(15c) \quad \lim_{\eta \rightarrow \infty} F^n(\eta, \theta, \zeta) = 0 , \quad n = 0, 1, \dots .$$

An analysis, similar to the preceding, applied to the boundary at "infinity" yields a corresponding formulation of that boundary layer problem. We do not exhibit this formulation.

[†] In obtaining (15c) we assume that the terms in f^n which vanish as $\eta \rightarrow \infty$ do so faster than any negative power of η .

5. Analysis of the Boundary Layer Problems.

For each n Eqs. (12-15) and (17) separate into two distinct systems which we call Problem P^n and Problem T^n .

Problem P^n is concerned with the coefficients f_r^n , f_z^n , f_θ^n and f_{rz}^n and involves the first two of (12), the first four of (13), (15) and (17) for these coefficients and the first two of each of (14). Problem T^n is concerned with the remaining two coefficients $f_{r\theta}^n$ and $f_{\theta z}^n$ and the remaining equations in (12-15) and (17).

We shall associate with P^n a "stress function", $\phi^n(\eta, \theta, \zeta)$, which may be the solution of the following boundary value problem on the semi-infinite strip, $|\zeta| \leq 1$, $\eta \geq 0$, and fixed θ :

$$\nabla^4 \phi^n = 0 ;$$

$$(18) \quad \phi_{\eta\eta}^n(\eta, \theta, \pm 1) = \phi_{\eta\zeta}^n(\eta, \theta, \pm 1) = 0; \quad \lim_{\eta \rightarrow \infty} [\phi_{\zeta\zeta}^n, \phi_{\eta\zeta}^n] = 0 ;$$

$$\phi_{\zeta\zeta}^n(0, \theta, \zeta) = \beta^n(\theta, \zeta) , \quad \phi_{-\eta\zeta}^n(0, \theta, \zeta) = 0 ,$$

where $\beta^n(\theta, \zeta) = \beta^n(\theta, -\zeta)$. It can be shown by elementary means that if $\phi_{\zeta\zeta}^n$, $\phi_{\eta\zeta}^n$ and $\phi_{\eta\eta}^n$ are uniformly continuous functions of ζ and if $\phi_{\zeta\zeta}^n$, $\phi_{\eta\eta}^n$ and $\phi_{\eta\zeta}^n$ are single-valued functions then,

$$(19) \quad \int_{-1}^1 \beta^n(\theta, \zeta) d\zeta = 0 \quad \text{for all } \theta .$$

Employing Problem T^0 , we can show that a solution of P^0 is given by

$$(20a) \quad F_r^0 = \phi_{\zeta\zeta}^0, \quad F_z^0 = \phi_{\eta\eta}^0, \quad F_{rz}^0 = -\phi_{\eta\zeta}^0, \quad F_\theta^0 = v \nabla^2 \phi^0$$

if ϕ^0 is the solution of (18) with $n = 0$ and

$$(20b) \quad \beta^0(\theta, \zeta) = -S_r^0(0, \theta) .$$

The first boundary condition on $\xi = 0$ for Problem I^0 is obtained from (20b) and (19) as,

$$(21) \quad S_r^0(0, \theta) = 0 .$$

Using Problem I^0 it follows that

$$(22a) \quad F_{r\theta}^0 = -\psi_{,\zeta}^0 , \quad F_{\theta z}^0 = \psi_{,\eta}^0$$

is a solution of Problem T^0 if $\psi^0(\eta, \theta, \zeta)$ is the solution of,

$$\nabla^2 \psi^n = 0 ,$$

(23)

$$\psi_{,\eta}^n(\eta, \theta, \pm 1) = 0 , \quad \lim_{\eta \rightarrow \infty} [\psi_{,\zeta}^n, \psi_{,\eta}^n] = 0 , \quad \psi_{,\zeta}^n(0, \theta, \zeta) = g^n(\theta, \zeta)$$

with $n = 0$ where,

$$(22b) \quad g^0(\theta, \zeta) = -S_{r\theta}^0(0, \theta) .$$

Single valuedness of the solution of (23) yields from (22b) the second boundary condition on $\xi = 0$ for I^0 as,

$$(24) \quad S_{r\theta}^0(0, \theta) = 0 .$$

From (18) and (20-24) it follows that

$$(25) \quad F^0(\eta, \theta, \zeta) \equiv 0 .$$

In a similar manner it may be shown from an analysis of Problems P^1 and T^1 that

$$(26) \quad F^1(\eta, \theta, \xi) \equiv 0$$

and that the boundary conditions on $\xi = 0$ for Problem I^1 are given by,

$$(27) \quad S_r^1(0, \theta) = S_{r\theta}^1(0, \theta) = 0 .$$

A solution of Problem P^2 is obtained from (18) with $n = 2$ and

$$(28a) \quad \beta^2(\theta, \xi) = -S_r^2(0, \theta) - \frac{1}{2} A^0(0, \theta) \xi^2$$

if

$$(28b) \quad F_r^2 = \phi_{,\xi\xi}^2, \quad F_z^2 = \phi_{,\eta\eta}^2, \quad F_{rz}^2 = -\phi_{,\eta\xi}^2, \quad F_\theta^2 = v \nabla^2 \phi^2 .$$

Equations (19) and (28a) yield the first boundary condition for Problem I^2 as,

$$(29) \quad S_r^2(0, \theta) = -\frac{1}{6} A^0(0, \theta)$$

where A^0 is obtained from (5) and (3a). Similarly we obtain a solution of Problem T^2 by setting

$$F_{r\theta}^2 = -\psi_{,\xi}^2, \quad F_{\theta z}^2 = \psi_{,\eta}^2$$

where $\psi^2(\eta, \theta, \xi)$ is a solution of (23) with $n = 2$ and

$$g^2(\theta, \xi) = S_{r\theta}^2(0, \theta) + \frac{1}{2} C^0(0, \theta) \xi^2 .$$

Single valuedness of the solution gives the second boundary condition for Problem I² as,

$$(30) \quad S_{r\theta}^2(0, \theta) = -\frac{1}{6} C^0(0, \theta) .$$

More accurate approximations of the stresses near the edge can be obtained by examining Problems Pⁿ and Tⁿ for n ≥ 3.

A corresponding analysis of the boundary layer at "infinity" yields the appropriate boundary conditions at infinity for the interior problems. For example, we can show that,

$$(31a) \quad S_r^0(\infty, \theta) = \frac{T}{2}(1 + \cos 2\theta) , \quad S_{r\theta}^0(\infty, \theta) = -\frac{T}{2} \sin 2\theta ,$$

$$(31b) \quad S_r^n(\infty, \theta) = S_{r\theta}^n(\infty, \theta) = 0 \quad \text{if } n = 1, 2 .$$

6. Solution of the Interior and Boundary Layer Problems.

Problem I⁰ (the plane stress theory) which consists of the differential equations (7) and (8a) with n = 0 and the boundary conditions (21), (24) and (31a) has the solution [1]:

$$(32) \quad \begin{aligned} \frac{2}{T} \sigma_r^0 &= \frac{2}{T} S_r^0 = 1 - \frac{1}{(1+\xi)^2} + \left[1 + \frac{3}{(1+\xi)^4} - \frac{4}{(1+\xi)^2} \right] \cos 2\theta , \\ \frac{2}{T} \sigma_\theta^0 &= \frac{2}{T} S_\theta^0 = 1 + \frac{1}{(1+\xi)^2} - \left[1 + \frac{3}{(1+\xi)^4} \right] \cos 2\theta , \\ \frac{2}{T} \sigma_{r\theta}^0 &= \frac{2}{T} S_{r\theta}^0 = - \left[1 + \frac{2}{(1+\xi)^2} - \frac{3}{(1+\xi)^4} \right] \sin 2\theta , \\ \sigma_{rz}^0 &= \sigma_{\theta z}^0 = \sigma_z^0 = 0 . \end{aligned}$$

Since the differential equations (7) and the boundary conditions (27) and (31b) are homogeneous we have for the solution of Problem I¹:

$$(33) \quad \sigma^1 \equiv 0 .$$

Employing (32) and (3) we obtain the solution of Problem I², which is given by (7) and (8b) with $n = 2$ and (29), (30) and (31b), as

$$(34a) \quad \sigma_r^2 = -\sigma_\theta^2 = G \cos 2\theta, \quad \sigma_{r\theta}^2 = G \sin 2\theta,$$

$$\sigma_{rz}^2 = \sigma_{\theta z}^2 = \sigma_z^2 = 0 ,$$

where

$$(34b) \quad G(\xi, \zeta) = \frac{2vT}{(1+v)} \frac{(1-3\xi^2)}{(1+\xi)^4} .$$

To obtain a solution of Problem P² we must solve the boundary value problem (18) with $n = 2$ where β^2 is given by (28a), (29), (3a) and (32). Since an exact solution of this problem is unknown we employ the "approximate" solution given by Horvay [13]:

$$(35a) \quad \begin{aligned} F_r^2 &= H \frac{1}{\beta} e^{-a\eta} (\cos b\eta + \frac{a}{b} \sin b\eta) (3\xi^2 - 1) \cos 2\theta , \\ F_z^2 &= H \left(\frac{a^2 + b^2}{12} \right) e^{-a\eta} (-\cos b\eta + \frac{a}{b} \sin b\eta) (1 - \xi^2)^2 \cos 2\theta , \\ F_{rz}^2 &= H \left(\frac{a^2 + b^2}{3b} \right) e^{-a\eta} \sin b\eta (\xi - \xi^3) \cos 2\theta , \end{aligned}$$

$$F_\theta^2 = v(F_r^2 + F_z^2) ,$$

where

the first time, the author has been able to find a single example of a *labeled* *graph* in the literature.

It is the purpose of this paper to introduce the concept of a labeled graph and to show how it can be used to solve some problems in graph theory.

The paper is organized as follows: In section 2 we give a brief introduction to the concept of a labeled graph. In section 3 we discuss some properties of labeled graphs. In section 4 we give some applications of labeled graphs to graph theory.

We would like to thank the referee for his valuable suggestions and comments which have greatly improved the presentation of this paper.

Received by the editors January 1, 1968; revised April 1, 1968.

© 1969 by the American Mathematical Society. Reprinted with permission from *Journal of Graph Theory*, Vol. 3, No. 1, pp. 1-10, © 1969 by the American Mathematical Society.

Reverts to public domain January 1, 1994.

$$(35b) \quad H = \frac{6vT}{1+v} , \quad a = 2.075 , \quad b = 1.143 .$$

An integral representation can be given for the solution of Problem T² since Green's function for the boundary value problem (23) is known [14]. However, we prefer to use the infinite series representation which yields for the solution of T²:

$$F_{r\theta}^2 = \sum_{n=1}^{\infty} K_n \cos n\pi\xi e^{-n\pi\xi} \sin 2\theta ,$$

(36a)

$$F_{\theta z}^2 = \sum_{n=1}^{\infty} K_n \sin n\pi\xi e^{-n\pi\xi} \sin 2\theta ,$$

where,

$$(36b) \quad K_n = \frac{4H}{\pi^2} \frac{(-1)^n}{n^2} .$$

As in [3] we define the stresses, $\sigma^{(N)}$, of the N-th approximation to the exact theory as:

$$(37) \quad \sigma^{(N)}(\xi, \theta, \zeta; \varepsilon) = \sum_{n=0}^N [\sigma^n(\xi, \theta, \zeta) + F^n(\xi/\varepsilon, \theta, \zeta)] \varepsilon^n .$$

It follows from this definition, (25), (26) and (33) that $\sigma^{(0)}$ and $\sigma^{(1)}$ coincide with the plane stress solution. For the second approximation to the exact theory we obtain from (37):

$$\begin{aligned}
 \sigma_r^{(2)}(\xi, \theta, \zeta; \varepsilon) &= S_r^0(\xi, \theta) + [\sigma_r^2(\xi, \theta, \zeta) + F_r^2(\xi/\varepsilon, \theta, \zeta)]\varepsilon^2, \\
 \sigma_\theta^{(2)}(\xi, \theta, \zeta; \varepsilon) &= S_\theta^0(\xi, \theta) + [\sigma_\theta^2(\xi, \theta, \zeta) + F_\theta^2(\xi/\varepsilon, \theta, \zeta)]\varepsilon^2, \\
 \tau_{r\theta}^{(2)}(\xi, \theta, \zeta; \varepsilon) &= S_{r\theta}^0(\xi, \theta) + [\sigma_{r\theta}^2(\xi, \theta, \zeta) + F_{r\theta}^2(\xi/\varepsilon, \theta, \zeta)]\varepsilon^2, \\
 (38) \quad \sigma_{rz}^{(2)}(\xi, \theta, \zeta; \varepsilon) &= F_{rz}^2(\xi/\varepsilon, \theta, \zeta)\varepsilon^2, \\
 \sigma_{\theta z}^{(2)}(\xi, \theta, \zeta; \varepsilon) &= F_{\theta z}^2(\xi/\varepsilon, \theta, \zeta)\varepsilon^2, \\
 \sigma_z^{(2)}(\xi, \theta, \zeta; \varepsilon) &= F_z^2(\xi/\varepsilon, \theta, \zeta)\varepsilon^2.
 \end{aligned}$$

Here S^0 , σ^2 and F^2 are given in (32) and (34-36).

7. Presentation of Results.

We define the quantity:

$$(39) \quad \hat{\sigma}_\theta^{(2)}(\xi, \zeta; \varepsilon) = \frac{\frac{\sigma_\theta^{(2)}(\xi, \theta, \zeta; \varepsilon)}{T} - \frac{1}{2} \left[1 + \frac{1}{(1+\xi)^2} \right]}{\cos 2\theta}.$$

In Fig. 1, $\hat{\sigma}_\theta^{(2)}(0, 0; \varepsilon)$ is illustrated as a function of ε with varying Poisson's ratio. For $\varepsilon = 0$ we obtain the plane stress result which gives a stress-concentration factor of two for $\hat{\sigma}_\theta^{(2)}$. Although the stress-concentration factor increases with ε , for "small" ε , it is only a small percentage of the plane stress solution. For example, with $\varepsilon = .2$ and $\nu = 1/4$ our results indicate only a 1.25% increase over the plane stress result. Thus, for "small" ε the plane stress solution apparently yields

accurate but unconservative results. The circles in Fig. 1 represent the results of Alblas [10] ($\nu = 1/4$) obtained from a formal infinite series solution of the exact theory. The dotted curve gives the "asymptotic solution" of Alblas.

Figure 2 reveals the behavior of $\hat{\sigma}_\theta^{(2)}(0, \pm 1; \varepsilon)$ as a function of ε . The curve obtained from the "asymptotic solution" of Alblas ($\nu = 1/4$ and $\varepsilon \leq 1/4$) coincides with our curve. In Fig. 3 the variation through the thickness of $\hat{\sigma}^{(2)}$ is indicated at the edge of the hole for $\nu = 1/4$ and varying ε . Since $\hat{\sigma}_\theta^{(2)}$ assumes its maximum on the middle surface, these results are of some importance in experiments where measurements are taken on the faces of the plate.

In the remaining three figures we illustrate "boundary layer behaviors" for some of the stresses. In Fig. 4 the variation with ξ of the middle surface values of $\hat{\sigma}_\theta^{(2)}$ is given for $\nu = 1/4$ and two values of ε . The ξ -variation of $\hat{\sigma}_\theta^{(2)}$ is relatively insensitive to changes in ε especially for $\varepsilon > .2$. Using the plane stress solution we define $\hat{\sigma}_\theta^{(0)}$ as

$$(40) \quad \hat{\sigma}_\theta^{(0)} = \frac{\frac{S_\theta^0}{T} - \frac{1}{2} \left[1 + \frac{1}{(1+\xi)^2} \right]}{\cos 2\theta} .$$

This quantity is in close agreement with $\hat{\sigma}_\theta^{(2)}(\xi, 0; \varepsilon)$ for $\varepsilon = 1/10$ and is not shown in Fig. 4.

Figure 5 shows the ξ -variation of the scaled thickness shear stress,

$$(41) \quad \hat{\sigma}_{z\theta}^{(2)}(\xi, \zeta; \varepsilon) = \left(\frac{100\pi^2(1+\nu)}{24\nu T \sin 2\theta} \right) \sigma_{z\theta}^{(2)}(\xi, \theta, \zeta; \varepsilon)$$

for $\zeta = 1/2$. We observe that for $\varepsilon = 1/4$, $\hat{\sigma}_{z\theta}^{(2)}$ at $\xi = .1$ is only 30 % of its maximum value, while at $\xi = .2$, $\hat{\sigma}_{z\theta}^{(2)}$ is 8.5 % of its maximum value. If we define the "boundary layer thickness", ξ^* , as that value of ξ at which $\hat{\sigma}_{z\theta}^{(2)}$ is a small percentage, say 5 %, of its maximum value then the results illustrated in Fig. 5 give

$$\xi^* \approx \varepsilon.$$

In the remaining graph, Fig. 6, the ξ variation of $\hat{\sigma}_{r\theta}^{(2)}$ is given on the face of the plate for $\nu = 1/4$. Here $\hat{\sigma}_{r\theta}^{(2)}$ is defined as,

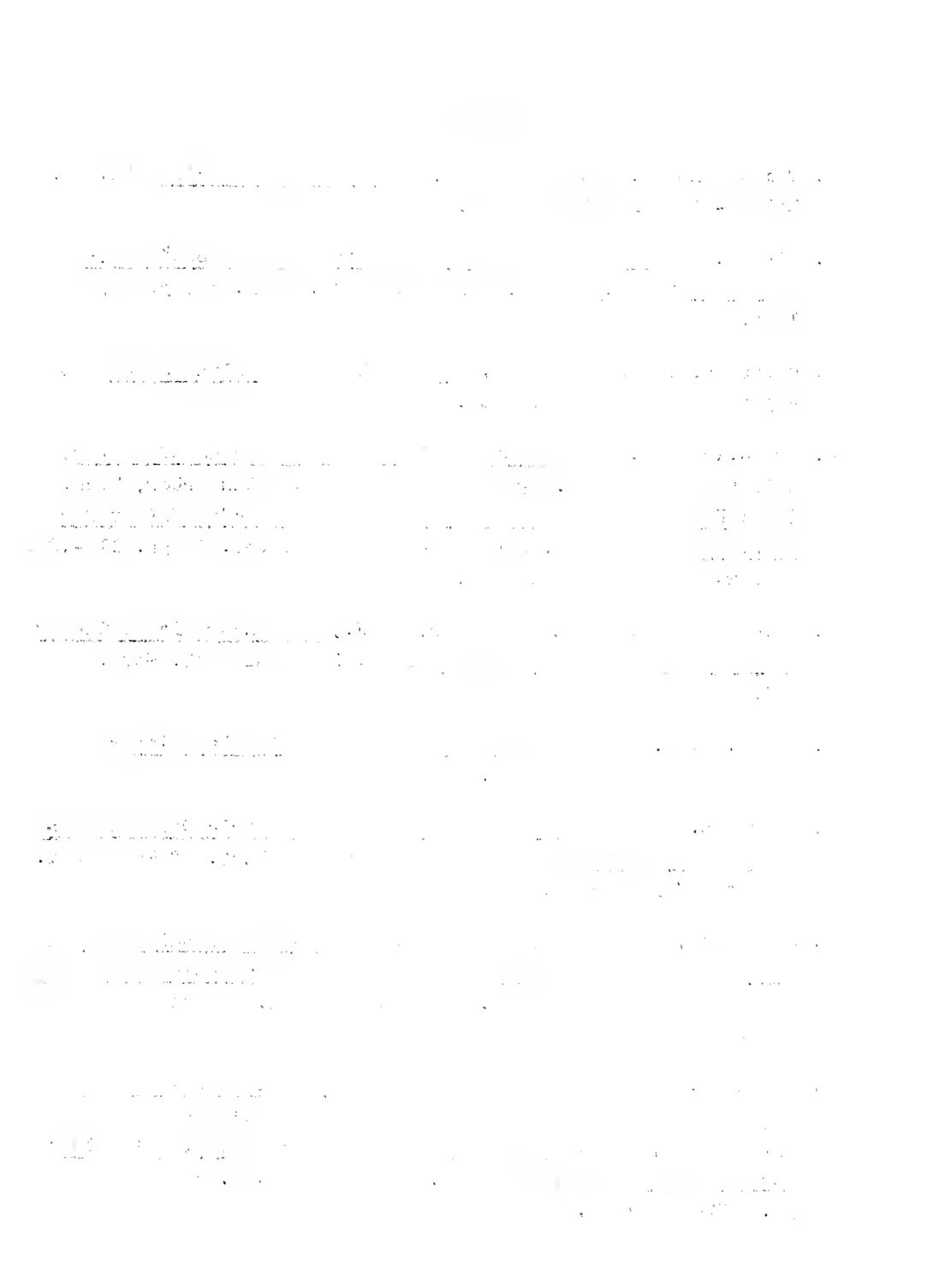
$$(42) \quad \hat{\sigma}_{r\theta}^{(2)}(\xi, \zeta; \varepsilon) = 125 \left\{ \frac{2\sigma_{r\theta}^{(2)}(\xi, \theta, \zeta; \varepsilon)}{T \sin 2\theta} + \left[1 + \frac{2}{(1+\xi)^2} - \frac{3}{(1+\xi)^4} \right] \right\}.$$

For the plane stress solution, $\varepsilon = 0$, the corresponding $\hat{\sigma}_{r\theta}^{(2)}$ coincides with the ξ axis.

We wish to emphasize that more accurate approximations to the exact solution can be obtained by extending our calculations to terms of order three and greater. Additional accuracy in our solution (38) could be obtained if the exact solution to Problem P², rather than (35), were available. Since (36) provides the exact solution to Problem T², the stresses $\sigma_{r\theta}^{(2)}$ and $\sigma_{\theta z}^{(2)}$ are most likely given with greater precision than the other stresses in (38).

References

1. Timoshenko, S. and Goodier, J., Theory of Elasticity, 2nd Ed., McGraw-Hill, New York, 1951.
2. Kirsch, G., Die Theorie der Elastizität und die Bedürfnisse der Festigkeitslehre, Z. Ver. Deut. Ing., Vol. 42, p. 797, 1898.
3. Reiss, E. L. and Locke, S., On the Theory of Plane Stress, to appear in the Q. Appl. Math.
4. Friedrichs, K. O., The Edge Effect in the Bending of Plates, Reissner Anniv. Vol., pp. 197-210, Edwards, Ann Arbor, Mich., 1949; Kirchoff's Boundary Conditions and the Edge Effect for Elastic Plates, Proc. Sym. in Appl. Math., Vol. 3, pp. 117-124, McGraw-Hill, New York, 1950.
5. Friedrichs, K. O. and Dressler, R. F., A Boundary Layer Theory for Elastic Bending of Plates, Comm. Pure and Appl. Math., Vol. 14, 1961.
6. Reiss, E. L., Symmetric Bending of Thick Circular Plates, submitted for publication.
7. Reiss, E. L., A Theory for the Small Unsymmetric Deformations of Cylindrical Shells, Report IMM-NYU 274, Inst. of Math. Sci., New York Univ., 1960.
8. Sternberg, E. and Sadowsky, M. A., Three-Dimensional Solution for the Stress Concentration Around a Circular Hole in a Plate of Arbitrary Thickness, J. Appl. Mech., Vol. 16, pp. 27-38, 1949.
9. Green, A. E., Three-Dimensional Stress Systems in Isotropic Plates, Phil. Trans. Roy. Soc. London (A), Vol. 240, pp. 561-597, 1948; The Elastic Equilibrium of Isotropic Plates and Cylinders, Proc. Roy. Soc. London (A), Vol. 195, pp. 533-552, 1949.



10. Alblas, J. B., Theorie Van De Driedimensionale Spanningstoestand in een Doorboorde Plaat, Doctoral Dissertation, Delft, Holland, 1957.
11. Reissner, E., On the Calculation of Three-Dimensional Corrections for the Two-Dimensional Theory of Plane Stress, Proc. 15th Eastern Photoelasticity Conf., pp. 23-31, Boston, 1942.
12. Friedrichs, K. O., Asymptotic Phenomena in Mathematical Physics, Bull. A.M.S., Vol. 61, pp. 485-504, 1955.
13. Horvay, G., The End Problem of Rectangular Strips, J. Appl. Mech., Vol. 20, pp. 87-94, 1953.
14. Friedrichs, K. O., Methods of Mathematical Physics, Lecture Notes, Inst. of Math. Sci., New York Univ., 1954.

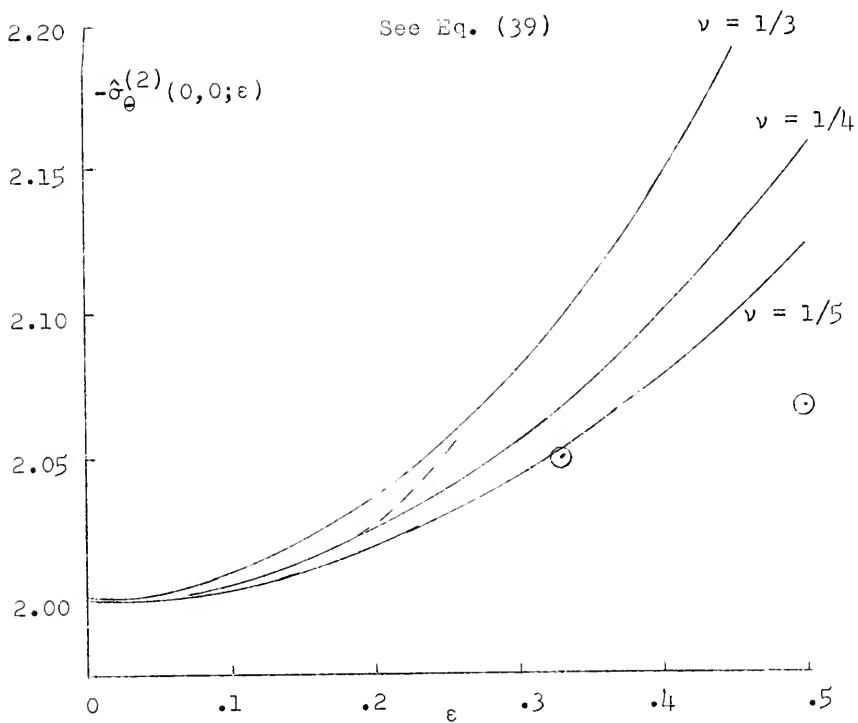


Figure 1: Variation with ε and ν of the middle plane values of $\hat{\sigma}_\theta^{(2)}$ at the edge of the hole.

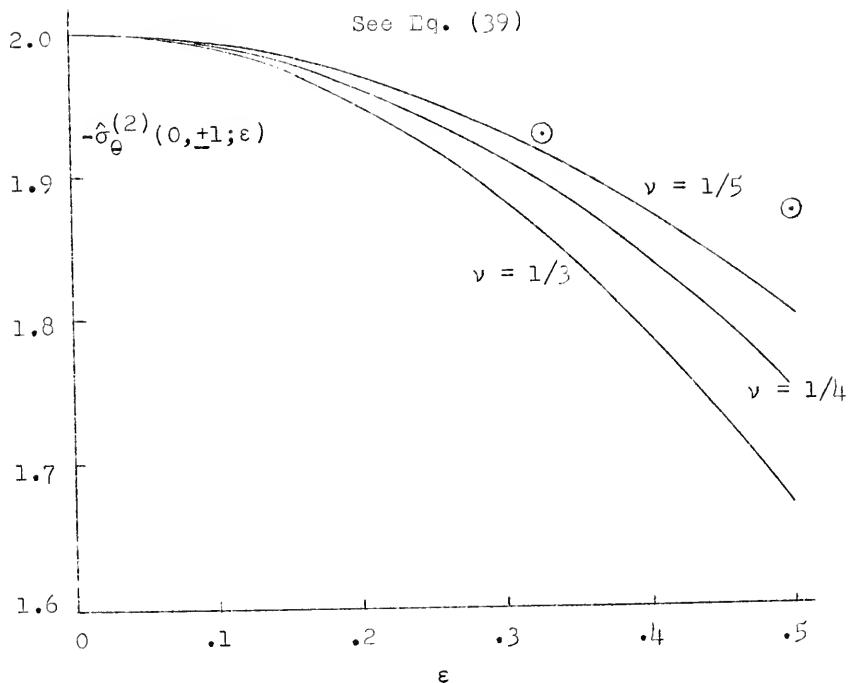


Figure 2: Variation with ϵ and ν of the face plane values of $\hat{\sigma}_\theta^{(2)}$ at the edge of the hole.

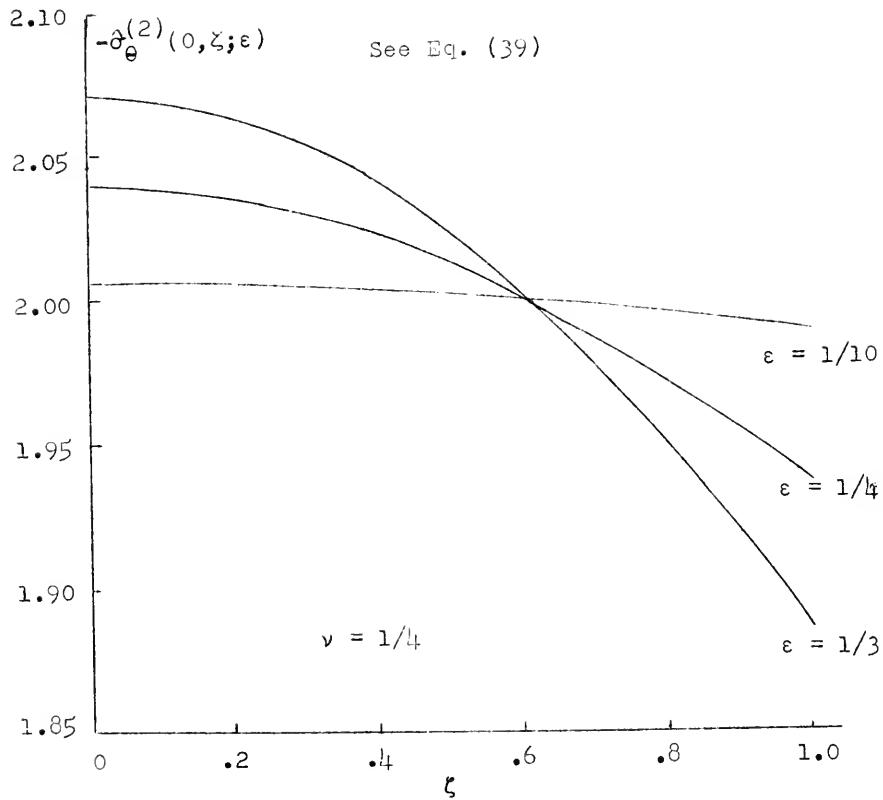


Figure 3: Variation through the thickness of $\hat{\sigma}_\theta^{(2)}$ at the edge of the hole for $\nu = 1/4$.

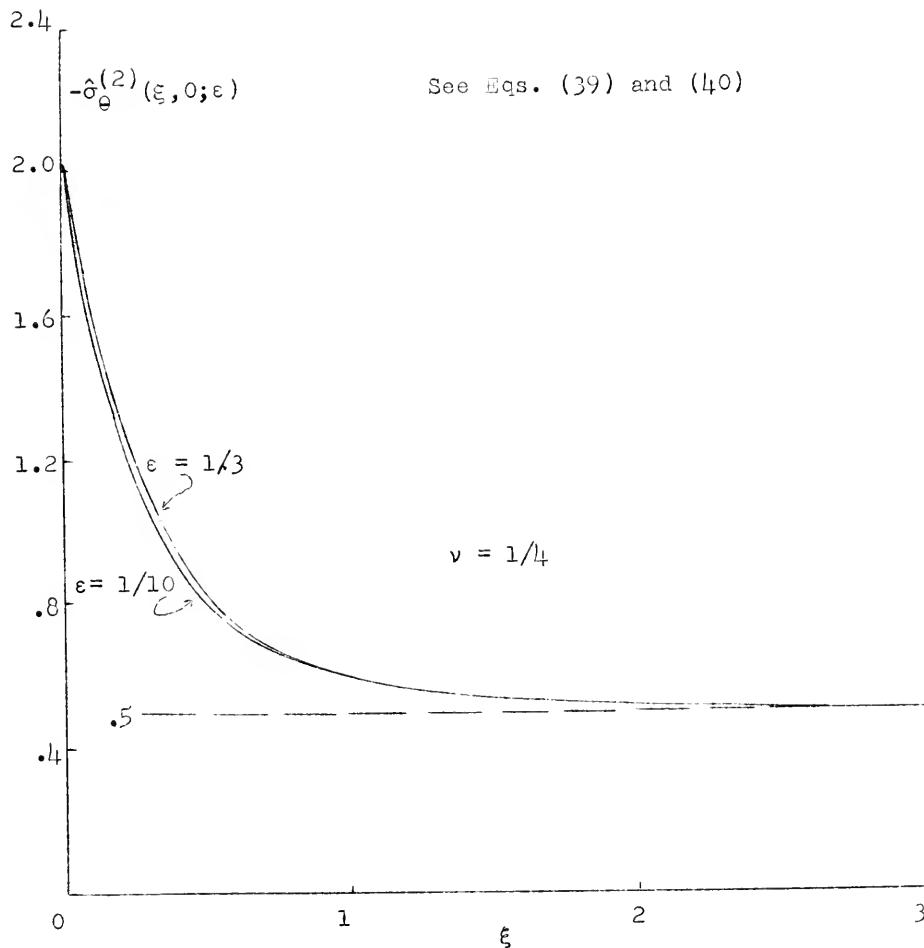
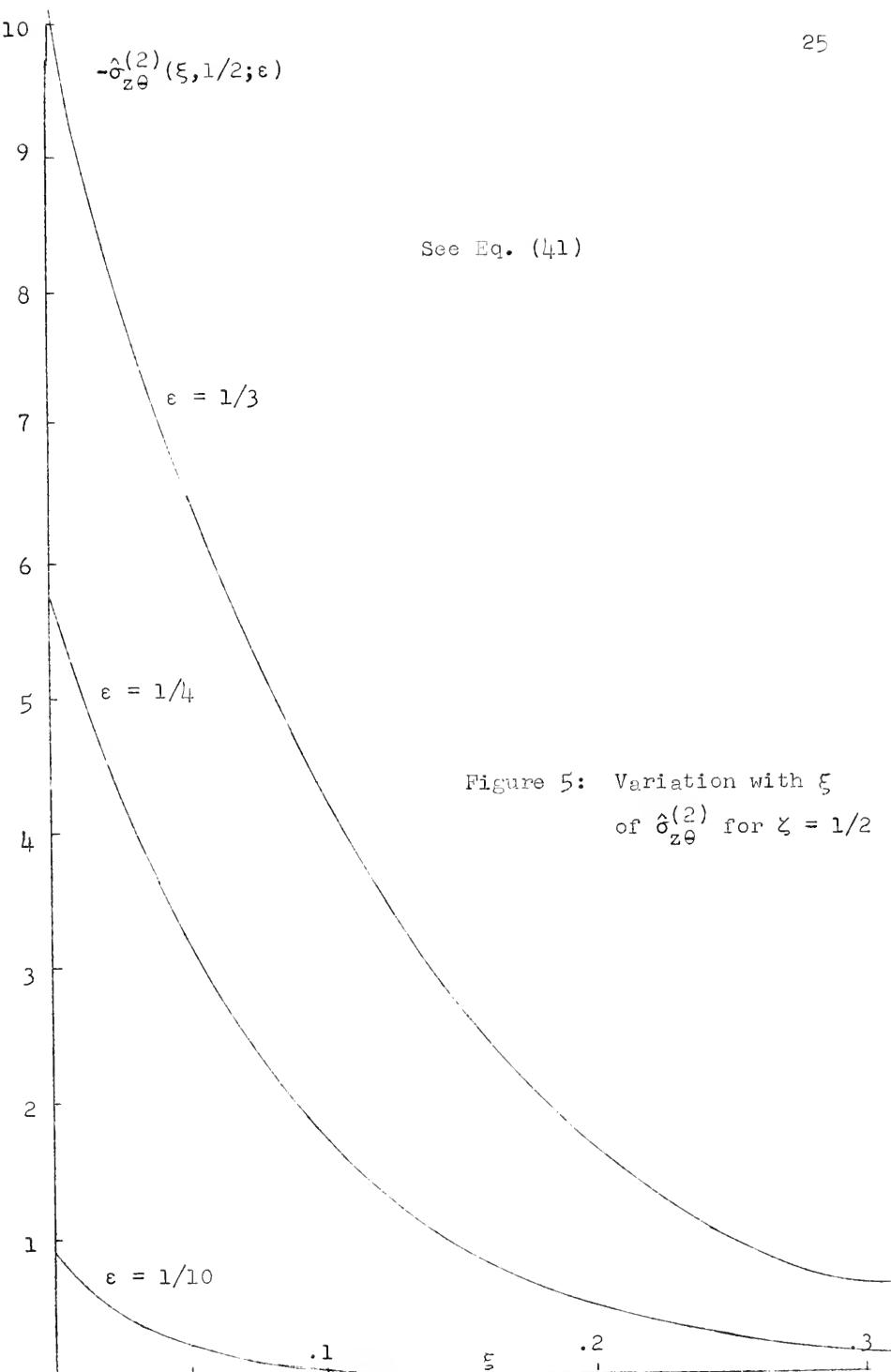


Figure 4: Variation with ξ of the middle plane values of $\hat{\theta}_\theta^{(2)}$ for $\nu = 1/4$.



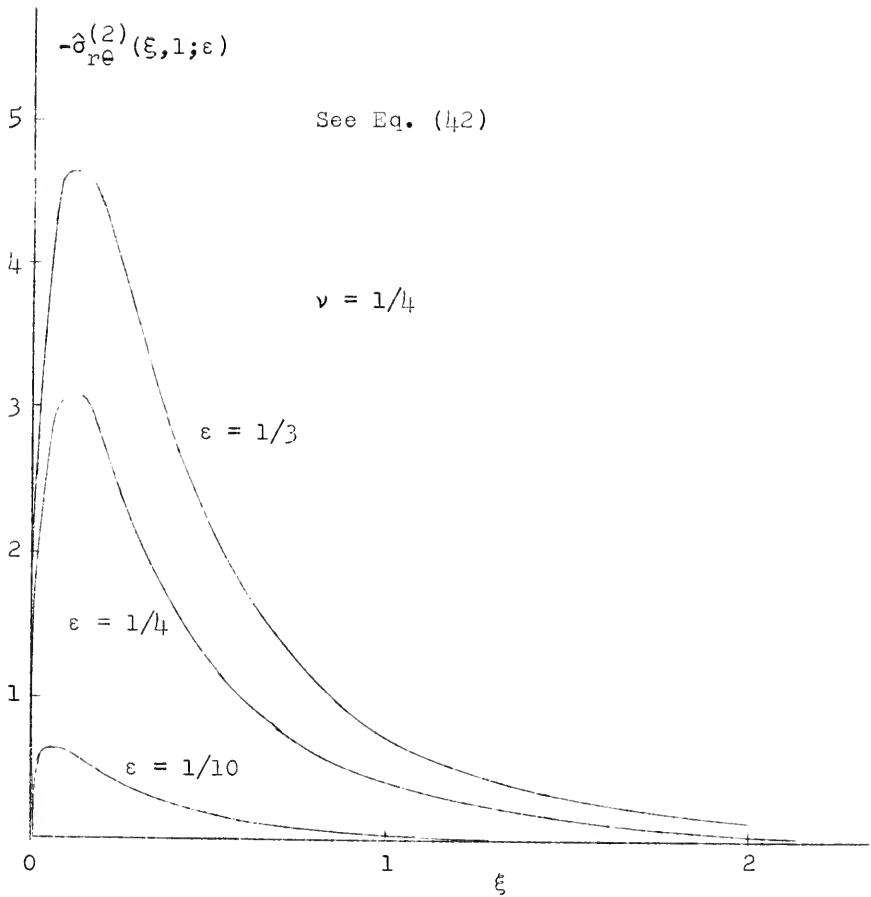


Figure 6: Variation with ξ of $\hat{\sigma}_{r\theta}^{(2)}$ for $\zeta = 1$.

DISTRIBUTION LIST UNCLASSIFIED TECHNICAL REPORTS
issued under Contract Nonr-285(42)

| | |
|------------------------------------|--|
| Chief of Nav. Res. | Chief of Staff, Dept. of Army |
| Dept. of Navy, Washington 25, D.C. | Washington 25, D.C. |
| Attn: Code 438 | (2) Attn: Dev. Br. (R&D Div.) (1) |
| | Res. Br. (R&D Div.) (1) |
| CO, Office of Nav. Res. | Spec. Weapons Br. (R&D) (1) |
| Br. Office, 495 Summer St. | |
| Boston 10, Massachusetts | (1) Office of Chief of Engineers |
| CO, Office of Nav. Res. | Dept. of Army, Washington 25, D.C. |
| Br. Office, J. Crerar Library | Attn: ENG-HL Lib. Br. Adm. |
| 86 E. Randolph St. | Ser. Div. (1) |
| Chicago 11, Illinois | ENG-WD Plan.Div.Civ.Wks (1) |
| CO, Office of Nav. Res. | ENG-EB Port. Constr. Br., Eng. Div.Mil.Cons. (1) |
| Br. Office, 346 Broadway | ENG-EA Struc. Br. Eng. Div. Mil. Constr. (1) |
| New York 13, N. Y. | ENG-NB Spec. Engr. Br., Eng. R&D Div. (1) |
| CO, Office of Nav. Res. | |
| Br. Office, 1030 E. Green St. | |
| Pasadena, California | (1) CO. Engin. Res. Dev. Lab. |
| CO, Office of Nav. Res. | Fort Belvoir, Virginia (1) |
| Br. Office, 1000 Geary St. | |
| San Francisco, California | (1) Office of Chief of Ordnance |
| CO, Office of Nav. Res. | Dept. of Army, Washington 25, D.C. |
| Br. Office, Navy 100, Fleet P.O. | Attn: Res. and Mat. Br., (Ord. R&D Div.) (1) |
| New York, N. Y. | (25) |
| Dir., Nav. Res. Labs. | |
| Washington 25, D.C. | |
| Attn: Tech. Info. Officer | (6) Office of Chief Signal Officer |
| Code 6200 | Dept. of Army, Washington 25, D.C. |
| Code 6205 | Attn: Engin. and Techn. Div. (1) |
| Code 6250 | |
| Code 6260 | (1) CO, Watertown Arsenal |
| ASTIA, Arlington Hall Station | Watertown, Massachusetts (1) |
| Arlington 12, Virginia | Attn: Lab. Div. |
| Office of Techn. Services | |
| Dept. of Commerce | |
| Washington 25, D. C. | (1) CO, Frankford Arsenal |
| Dir. of Def. Res. and Engin. | Bridesburg Station |
| The Pentagon, Washington 25, D.C. | Philadelphia 37, Penna. (1) |
| Attn: Techn. Library | Attn: Lab. Div. |
| Chief, Armed Forces Special | |
| Weapons Project | |
| The Pentagon, Washington 25, D.C. | |
| Attn: Techn. Info. Div. | (2) Office of Ordnance Research |
| Weapons Effects Div. | 2127 Myrtle Dr., Duke Station (1) |
| Spec. Field Projects | Durham, North Carolina (1) |
| Blast and Shock Br. | Attn: Div. of Engin. Sci. (1) |
| Office of Secy. of the Army | |
| The Pentagon, Washington 25, D.C. | |
| Attn: Army Library | (1) CO, Squier Signal Lab. |
| | Fort Monmouth, N. J. (1) |
| | Attn: Comp. and Mat. Br. (1) |
| | Chief of Naval Operations |
| | Dept. of Navy, Washington 25, D.C. (1) |
| | Attn: Op 91 (1) |
| | Op 03EG (1) |
| | Commandant, Marine Corps |
| | Headquarters, USMC |
| | Washington 25, D. C. (1) |

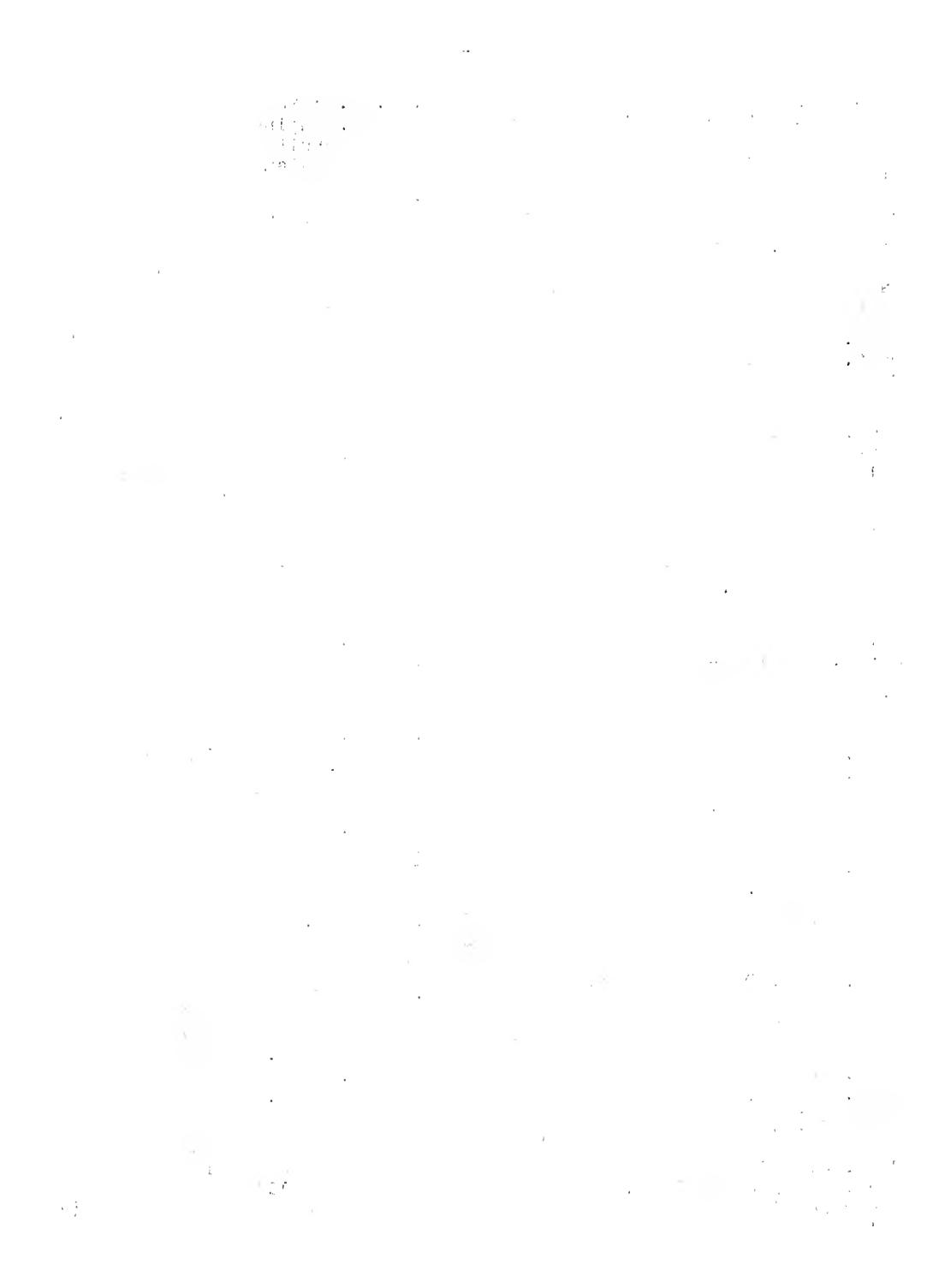
| | | | |
|--|-----|--|-----|
| Chief, Bureau of Ships Dept. of Navy Washington 25, D. C. | | Commanding Officer and Director David Taylor Model Basin Washington 7, D. C. | |
| Attn: Code 106 | (1) | Attn: Code 140 | (1) |
| Code 312 | (5) | Code 600 | (1) |
| Code 320 | (1) | Code 700 | (1) |
| Code 370 | (1) | Code 720 | (1) |
| Code 375 | (1) | Code 725 | (1) |
| Code 420 | (1) | Code 731 | (1) |
| Code 421 | (1) | Code 740 | (2) |
| Code 423 | (2) | CO, U.S. Naval Ordnance Lab. White Oak, Maryland | |
| Code 425 | (1) | Attn: Techn. Library | (2) |
| Code 440 | (1) | Techn. Eval. Dept. | (1) |
| Code 442 | (2) | Director, Materials Lab. | |
| Code 443 | (1) | N.Y. Naval Shipyard | |
| Code 525 | (1) | Brooklyn 1, N. Y. | (1) |
| Code 633 | (1) | CO, Portsmouth Naval Shipyard Portsmouth, New Hampshire | (2) |
| Chief, Bureau of Aeronautics Dept. of Navy Washington 25, D. C. | | CO, Mare Island Nav. Shipyard Vallejo, California | (2) |
| Attn: AE-4 | (1) | CO and Director U.S. Nav. Electron. Lab. | |
| AV-34 | (1) | San Diego 52, California | (1) |
| AD | (1) | Officer-in-Charge Nav. Civ. Engin. Res. | |
| AD-2 | (1) | and Eval. Lab. | |
| RS-7 | (1) | U.S. Nav. Constr. Battal. Center Port Hueneme, California | (2) |
| RS-8 | (1) | Dir., Nav. Air Experimental Sta. Nav. Air Mat. Center, Nav. Base | |
| SI | (1) | Philadelphia 12, Penna. | |
| AER-126 | (1) | Attn: Materials Lab. | (1) |
| Chief, Bureau of Ordnance Dept. of Navy Washington 25, D. C. | | Structures Lab. | (1) |
| Attn: Ad3 | (1) | Officer-in-Charge Underwater Explos. Res. Div. | |
| Re | (1) | Norfolk Naval Shipyard | |
| ReS | (1) | Portsmouth, Virginia | |
| ReU | (1) | Attn: Dr. A. H. Keil | (2) |
| ReS5 | (1) | CO, U.S. Nav. Proving Ground | |
| ReS1 | (1) | Dahlgren, Virginia | (1) |
| Ren | (1) | Supr. of Shipbuilding | |
| Spec. Proj. Office, Bur. Ord. Dept. of Navy Washington 25, D. C. | (2) | USN and Nav. Inspec. of Ordnance | |
| Attn: Missile Br. | | General Dynamics Corp., | |
| Chief, Bur. Yards and Docks Dept. of Navy Washington 25, D. C. | | Electr. Boat Div. | |
| Attn: Code D-202 | (1) | Groton, Connecticut | (1) |
| Code D-202.3 | (1) | Supr. of Shipbuilding | |
| Code 220 | (1) | USN and Nav. Inspec. of Ordnance | |
| Code D-222 | (1) | Newport News Shipbuilding | |
| Code D-410C | (1) | and Dry Dock Co. | |
| Code D-440 | (1) | Newport News, Virginia | (1) |
| Code D-500 | (1) | | |



| | | | |
|--|------------|---|-------------------|
| Supr. of Shipbuilding USN and Nav. Inspec. of Ordnance Ingalls Shipbuilding Corp. Pascagoula, Mississippi | (1) | U.S. Atomic Energy Commission Washington 25, D. C. Attn: Dir. of Research | (2) |
| CO, U.S. Nav. Admin. Unit MIT, Cambridge 39, Mass. | (1) | Dir., Nat. Bur. of Standards Washington 25, D. C. Attn: Div. of Mechanics Engin. Mech. Sect. Aircraft Structures | (1) (1) (1) |
| Officer-in-Charge Postgrad. School for Naval Officers Webb Inst. of Nav. Arch. Crescent Beach Rd. Glen Cove, L.I., N.Y. | (1) | Comm., U.S. Coast Guard 1300 E St., NW Washington 25, D. C. Attn: Chief, Test and Dev. Div | (1) |
| Supt., Nav. Gun Factory Washington 25, D. C. | (1) | U.S. Maritime Administration General Admin. Office Bldg. 441 G St., NW Washington 25, D. C. Attn: Chief, Div. Prelim. Design | (1) (1) |
| Comm., Nav. Ordnance Test Sta. China Lake, California Attn: Physics Div. Mechanics Div. | (1) (1) | Nat. Aero. and Space Admin. Langley Research Center Langley Field, Virginia Attn: Structures Div. | (2) |
| CO, Nav. Ordnance Test Sta. Underwater Ordnance Div. 3202 E. Foothill Blvd. Pasadena 8, California Attn: Struc. Div. | (1) | Nat. Aero. and Space Admin. 1512 H St., NW Washington 25, D. C. Attn: Loads and Struc. Div. | (2) |
| CO and Director U.S. Nav. Engin. Exp. Station Annapolis, Maryland | (1) | Director, Forest Prod. Lab. Madison, Wisconsin | (1) |
| Supt. U.S. Nav. Postgrad. School Monterey, California | (1) | Federal Aviation Agency Dept. of Commerce Washington 25, D. C. Attn: Chief, Air Engin. Div. Chief, Air. and Equip. Div. | (1) (1) |
| Comm. Marine Corps Schools Quantico, Virginia Attn: Dir., MC Dev. Center | (1) | National Science Foundation 1520 H St., NW Washington, D. C. | (1) |
| Comm. Gen., USAF Washington 25, D. C. Attn: Res. and Dev. Div. | (1) | National Academy of Sciences 2101 Constitution Ave., Washington 25, D. C. Attn: Dir., Comm. on Ships Struc. Design Exec. Secy., Comm. on Undersea Warfare | (1) (1) |
| CO, Air Material Command Wright-Patterson AFB, Ohio Attn: MCREX-B Structures Div. | (1) (1) | General Dynamics Corp. Electr. Boat Div. Groton, Connecticut | (1) |
| CO, USAF Inst. of Technology Wright-Patterson AFB, Ohio Attn: Chief, Appl. Mech. Group | (1) | Newport News Shipbuilding and Dry Dock Co. Newport News, Virginia | (1) |
| Director of Intelligence Headquarters, USAF Washington 25, D. C. Attn: PV Br. (Air Targ. Div) | (1) | | |



| | | | |
|---|-----|--|-----|
| Ingalls Shipbuilding Corp. Pascagoula, Mississippi | (1) | Prof. A. C. Eringen Dept. Aero. Engineering Purdue University Lafayette, Indiana | (1) |
| Prof. Lynn S. Beedle Fritz Engineering Lab. Lehigh University Bethlehem, Penna. | (1) | Prof. W. Flugge Dept. of Mech. Engineering Stanford, California | (1) |
| Prof. R. L. Bisplinghoff Dept. of Aero. Engineering Massachusetts Inst. of Techn. Cambridge 39, Massachusetts | (1) | Mr. M. Goland, VP and Dir. Southwest Research Institute 8500 Culebra Rd. San Antonio 6, Texas | (1) |
| Prof. H. H. Bleich Dept. of Civ. Engineering Columbia University New York 27, N. Y. | (1) | Prof. J. N. Goodier Dept. of Mech. Engineering Stanford University Stanford, California | (1) |
| Prof. B. A. Boley Dept. of Civ. Engineering Columbia University New York 27, N. Y. | (1) | Prof. L. E. Goodman Engineering Experimental Sta. University of Minnesota Minneapolis, Minnesota | (1) |
| Dr. John F. Brahtz Southern California Labs. Stanford Research Institute 820 Mission St. South Pasadena, California | (1) | Prof. M. Hetenyi The Technical Institute Northwestern University Evanston, Illinois | (1) |
| Dr. D. O. Brush Struc. Dept. 53-13 Lockheed Aircraft Corp. Missile Syst. Div. Sunnyvale, California | (1) | Prof. P. G. Hodge Dept. of Mechanics Illinois Inst. of Technology Chicago 16, Illinois | (1) |
| Prof. B. Budiansky Dept. of Mech. Engineering School Appl. Sciences Harvard University Cambridge 38, Massachusetts | (1) | Prof. N. J. Hoff, Head Div. Aeronautical Engineering Stanford University Stanford, California | (1) |
| Prof. Herbert Deresiewicz Dept. of Civ. Engineering Columbia University 632 W. 125th St. New York 27, N. Y. | (1) | Prof. W. H. Hopmann, II Dept. of Mechanics Rensselaer Polytechnic Inst. Troy, New York | (1) |
| Prof. D. C. Drucker, Chmn. Div. of Engineering Brown University Providence 12, Rhode Island | (1) | Prof. Bruce G. Johnston University of Michigan Ann Arbor, Michigan | (1) |
| Prof. John Duberg Dept. of Civ. Engineering University of Illinois Urbana, Illinois | (1) | Prof. J. Kempner Dept. of Aero. Engineering and Appl. Mechanics Polytechnic Inst. of Brooklyn 333 Jay St. Brooklyn 1, N. Y. | (1) |
| Prof. J. Erickson Mech. Engineering Dept. Johns Hopkins University Baltimore 18, Maryland | (1) | Prof. H. L. Langhaar Dept. of Theoretical and Applied Mechanics University of Illinois Urbana, Illinois | (1) |



| | | | |
|--|-----|---|-----|
| Prof. B. J. Lazan, Director Engineering Experimental Sts. University of Minnesota Minneapolis 14, Minnesota | (1) | Prof. W. Prager, Chmn. Phys. Sci. Council Brown University Providence 12, Rhode Island | (1) |
| Prof. E. H. Lee Div. of Appl. Mathematics Brown University Providence 12, Rhode Island | (1) | Prof. J. R. M. Radok Dept. of Aero Engineering and Appl. Mechanics Polytechnic Inst. of Brooklyn 333 Jay St. Brooklyn 1, N. Y. | (1) |
| Prof. George H. Lee, Dir. of Res. Rensselaer Polytechnic Inst. Troy, New York | (1) | Prof. E. L. Reiss Inst. of Mathematical Sciences New York University 4 Washington Place New York 3, N. Y. | (1) |
| Mr. S. Levy GE Electr. Research Lab. 3198 Chestnut St. Philadelphia, Penna. | (1) | Prof. E. Reissner Dept. of Mathematics Massachusetts Inst. of Technology Cambridge 39, Massachusetts | (1) |
| Prof. Paul Lieber Geology Department University of California Berkeley 4, California | (1) | Prof. M. A. Sadowsky Dept. of Mechanics Rensselaer Polytechnic Inst. Troy, New York | (1) |
| Prof. Joseph Marin, Head Dept. Engineering Mechanics College of Engin. and Arch. Pennsylvania State University University Park, Penna. | (1) | Prof. B. W. Shaffer Dept. of Mech. Engineering New York University University Heights New York 53, N. Y. | (1) |
| Prof. R. D. Mindlin Dept. of Civ. Engineering Columbia University 632 W. 125th St. New York 27, N. Y. | (1) | Prof. J. Stallmeyer Dept. of Civ. Engineering University of Illinois Urbana, Illinois | (1) |
| Prof. Paul M. Naghdi Building T-7 College of Engineering University of California Berkeley 4, California | (1) | Prof. Eli Sternberg Dept. of Mechanics Brown University Providence 12, Rhode Island | (1) |
| Prof. William A. Nash Dept. of Engineering Mechanics University of Florida Gainesville, Florida | (1) | Prof. T. Y. Thomas Grad. Inst. Math. and Mech. Indiana University Bloomington, Indiana | (1) |
| Prof. N. M. Newmark, Head Dept. of Civ. Engineering University of Illinois Urbana, Illinois | (1) | Prof. S. P. Timoshenko School of Engineering Stanford University Stanford, California | (1) |
| Prof. E. Orowan Dept. of Mech. Engineering Massachusetts Institute of Techn. Cambridge 39, Massachusetts | (1) | Prof. A. S. Velestos Dept. of Civ. Engineering University of Illinois Urbana, Illinois | (1) |
| Prof. Aris Phillips Dept. of Civ. Engineering 15 Prospect St. Yale University New Haven, Connecticut | (1) | | |

| | | | |
|--|-------------------|--|-----|
| Dr. E. Wenk Southwest Research Institute 8500 Culebra Rd. San Antonio, Texas | (1) | Legislative Reference Service Library of Congress Washington 25, D. C. Attn: Dr. E. Wenk | (1) |
| Prof. Dana Young Yale University New Haven, Connecticut | (1) | Dr. A. Ross Aircraft Nuclear Propulsion Dept. General Electric Co. Cincinnati 15, Ohio | (1) |
| Prof. R. A. Di Taranto Dept. of Mech. Engineering Drexel Institute 32nd and Chestnut Streets Philadelphia, Penna. | (1) | Dr. F. Lane General Applied Science Labs. Stewart and Merrick Avenues Westbury, L.I., N. Y. | (1) |
| Mr. H. K. Koopman, Secy. Welding Res. Council Engineering Foundation 29 W. 39th St. New York 18, N. Y. | (2) | Commander Edward Leonard Asst. Navy Representative MIT Lincoln Lab. Lexington 73, Massachusetts | (1) |
| Prof. Walter T. Daniels School of Engin. and Archit. Howard University Washington 1, D. C. | (1) | | |
| Comm., (Code 753) U.S. Naval Ordnance Test Sta. China Lake, California Attn: Techn. Library | (1) | | |
| Prof. J. E. Cermak Dept. of Civ. Engineering Colorado State University Fort Collins, Colorado | (1) | | |
| Prof. W. J. Hall Dept. of Civ. Engineering University of Illinois Urbana, Illinois | (1) | | |
| Dr. Hyman Serbin Design Integration Dept. Hughes Aircraft Co. Culver City, California | (1) | | |
| Commander Wright Air Development Center Wright-Patterson Air Force Base Dayton, Ohio Attn: Dynamics Branch Aircraft Lab. WCLSY | (1) (1) (1) | | |
| Commanding Officer USNNOEU Kirtland Air Force Base Albuquerque, New Mexico Attn: Code 20 (Dr. J.N. Brennan) | (1) | | |

MAR 29 1961 Date Due

18 63

NYSILL Spring Draft 7/20/70
PINT + the PINTS -



PRINTED IN U. S. A.

NYU
IMN-281
Reiss

c.l

Extension of a thick infinite
plate...

DATE DUE BORROWER'S NAME NUMBER

1 JAN 10 1968
Fayle - Sury Buf 7/30/70
[initials] [initials]

N. Y. U. Institute of
Mathematical Sciences
25 Waverly Place
New York 3, N. Y.

